**Amendments to the Specification:**

Please replace paragraph [0043] (previously amended) on pages 6-7 with the following replacement paragraph:

[0043] The movable plate 34 ensures the tension and the taking up of the length of chain which is not used. For this purpose, the arm 86 has pivoted in the direction of the arrow 84 into its extreme position, the first roller 88 being up and the second roller 90 being down. The chain passage hence follows a substantially serpentine path in the shape of a sharp or pronounced "S"-curve (see Fig. 2, path of chain 24 around rollers 88 and 90), such that the path of the chain around the first and second rollers 88 and 90 is in the shape of the letter "S". With the arm 86 pivoted as described in this paragraph and shown in Fig. 2, the path of the chain around the respective rollers 88 and 90 is as long as possible because the rollers 88 and 90 are positioned to take up the maximum possible amount of chain slack~~which corresponds to a long distance path~~.

Please replace paragraph [0046] on page 7 with the following replacement paragraph:

[0046] ~~The~~With the chain in this configuration, the tension arm 86~~, by a rotation inverse to~~ is rotated or pivoted in a direction opposite that produced by the return spring 82, such that the tension arm 86 is then carried into a position [[,]] in which the ~~S is scarcely pronounced and is lying~~ pathway of the chain around the first and second rollers 88 and 90 follows a relaxed "S"-curve (see Fig. 3, path of chain 24 around rollers 88 and 90) [[,]] ; i.e. the path of the chain around the first and second rollers 88 and 90 follows a serpentine path that loosely resembles a letter "S", but which is ~~which leads to a~~ *quasi* linear ~~chain course~~. In other words, with the arm 86 pivoted as described in this paragraph and shown in Fig. 3, the path of the chain around the respective rollers 88 and 90 is as short as possible because the rollers 88 and 90 are positioned to take up a minimum amount of chain slack, so the path of the chain through the rollers 88 and 90 is nearly linear or as linear as

possible. However, the chain path inevitably still exhibits some S-shaped curvature as a result of being carried over the rollers 88 and 90, which is what is meant by relaxed "S"-curve.

Please replace paragraphs [0048] and [0049] on page 7 with the following replacement paragraphs:

[0048] ~~In this configuration~~ With the tension arm 86 pivoted as described in the preceding two paragraphs, the ground clearance for the rollers 88 and 90 is even more improved than when the tension arm 86 is pivoted into its extreme position in the direction of arrow 84 in Fig. 2 as described above~~as compared with the preceding position~~.

[0049] Moreover, the ground clearance of the rollers 88 and 90 is considerably improved ~~in~~ for all positions of the tension arm 86 ~~in a considerable manner, and~~ compared to conventional dérailleur systems because the dérailleur (including tension arm 86 and rollers 88 and 90) is positioned in a space which essentially corresponds to the space required by the disks, whereby the ground clearance cannot be reduced. In other words, the ground clearance for the rollers 88 and 90, in all degrees of rotation of the tension arm 86, is about the same (or at least as good as) that of the disks A, B and C. It is understood by persons of ordinary skill in the dérailleur art that the ground clearance of a component refers to the distance from that component to the ground on which the bicycle is resting or being ridden. Obviously, the greater the ground clearance, the lesser the chance of fouling of the component from ground-based debris, or of contacting the ground while maneuvering the bicycle. This means 'improved' ground clearance refers to a greater distance between the component and the ground, whereas decreased ground clearance would mean less distance between the component and the ground.

Please replace paragraph [0053] on page 8 with the following replacement paragraph:

[0053] Thus, one has connected in Figure 5, which corresponds to the embodiment of Figure 1 with the same reference numerals for identical elements, a sector 102 of the roller fixed to the axis 80 supporting the tension arm ~~34~~ 86 by means of a cable 100, with one end 104 of the ~~ends of the~~ return spring 106 of the deformable parallelogram of the guiding/ dérailleur means.